

Analysis of Underground Coal Miner Heat Exposure in a Refuge Chamber

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Outline of Presentation

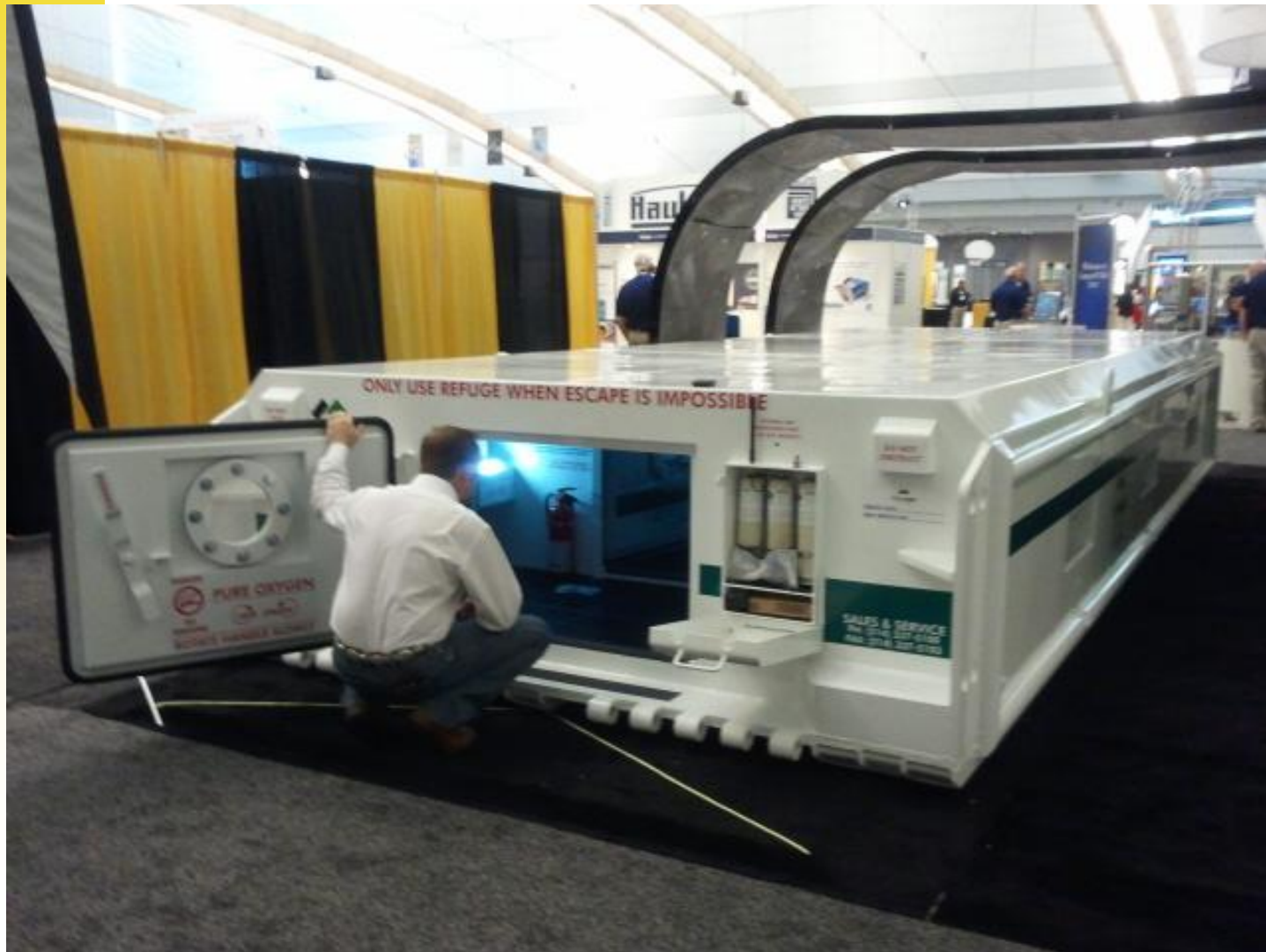
- 1) Introduction**
- 2) Previous Work**
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 - a) Impact due to Chamber Size**
 - b) Impact due to Chamber Structure**
 - c) Effect of External Temperature**
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Introduction

- ◆ **A refuge chamber is a prefabricated barricade used as a last resort in the event of a mine emergency**
- ◆ **Isolate miners from harmful gases and sustain life until miners can be rescued**
- ◆ **Two basic types: stationary and portable**
- ◆ **Collectively referred to as “refuge alternatives”**

Introduction

- ◆ **Have been required in underground metal/nonmetal mines since 1977**
- ◆ **U.S. coal mines have been a late adopter of refuge alternatives**
- ◆ **Sago Mine Disaster resulted in significant regulatory changes**
- ◆ **Must be located within 1,000 feet of the face and one hour's travel distance apart in outby areas**



Refuge Chamber (Longwall USA, <http://flic.kr/p/9Wr21S>)

Introduction

- ◆ **Refuge alternatives must meet performance standards per 30 CFR Part 7**
- ◆ **Some pertinent requirements:**
 - Internal apparent temperature must not exceed 35°C (95°F)
 - Provide a source of breathable air for 96 hours
 - Maintain average CO₂ concentrations below 1% vol.
 - All external electrical components must be intrinsically safe
 - Minimum requirements for floor space and unrestricted volume

Previous Work

- ◆ **Heat exposure is a safety hazard that must be considered in a small, enclosed space**
- ◆ **NIOSH has performed in-mine testing at Lake Lynn Laboratory**
- ◆ **Have tested a variety of refuge alternative types**
- ◆ **Experimental set up consisting of light bulbs, humidifiers, O₂ source, CO₂ scrubber**

Previous Work

- ◆ **Sensible heat output from light bulbs equivalent to 117 W per occupant (NASA standard)**
- ◆ **Used soda lime and lithium hydroxide CO₂ scrubbers**
- ◆ **42.8 W/person for soda lime**
- ◆ **52.2 W/person for lithium hydroxide**
- ◆ **Average external temperature of approximately 17°C (62°F)**

Previous Work

- ◆ **A steady state temperature was reached after 4-15 hours**
- ◆ **At steady state, air became saturated and moisture condensed on refuge alternative walls**
- ◆ **Some refuge alternatives exceeded the apparent temperature standard**

Model Design

- ◆ **In-mine testing has a number of deficiencies**
- ◆ **A 96-hour test using human occupants could be psychologically damaging**
- ◆ **Time consuming and expensive**
- ◆ **Each new refuge alternative design requires a series of new tests**
- ◆ **Simulation is a viable alternative to in-mine testing**

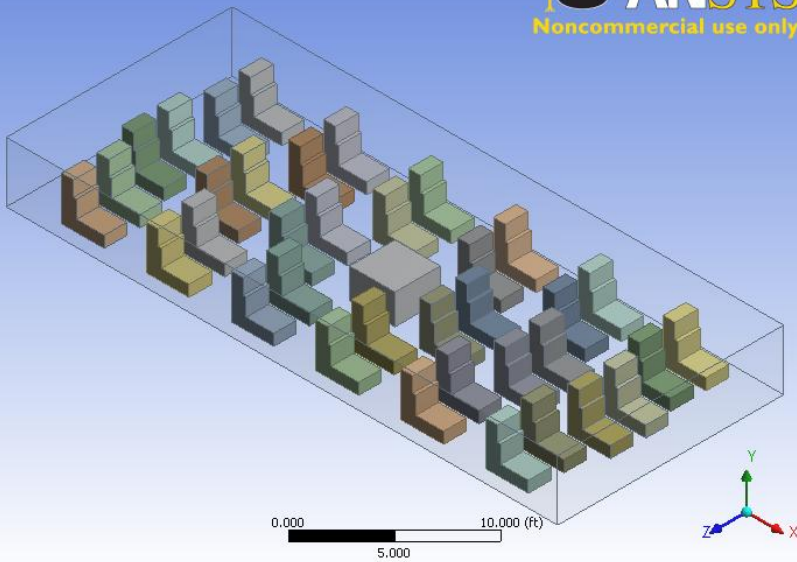
Model Design

- ◆ **Performed modeling in ANSYS Fluent of the temperature inside refuge chambers**
- ◆ **ANSYS Fluent is a computational fluid dynamics (CFD) program**
- ◆ **Generic chamber designed around minimum MSHA requirements for floor space and unrestricted volume**
- ◆ **Not based on any commercially available refuge chamber in service**

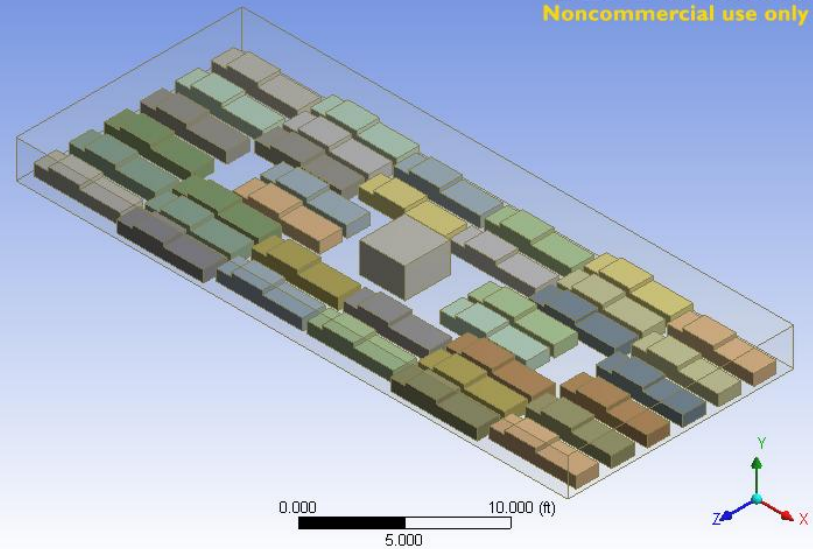
Model Design

Four steps in CFD modeling:

- 1) Create the Geometry**
- 2) Mesh the Geometry**
- 3) Fluid Dynamics/Heat Transfer Modeling**
- 4) Post-process the Results**



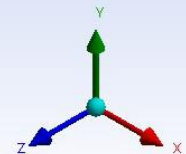
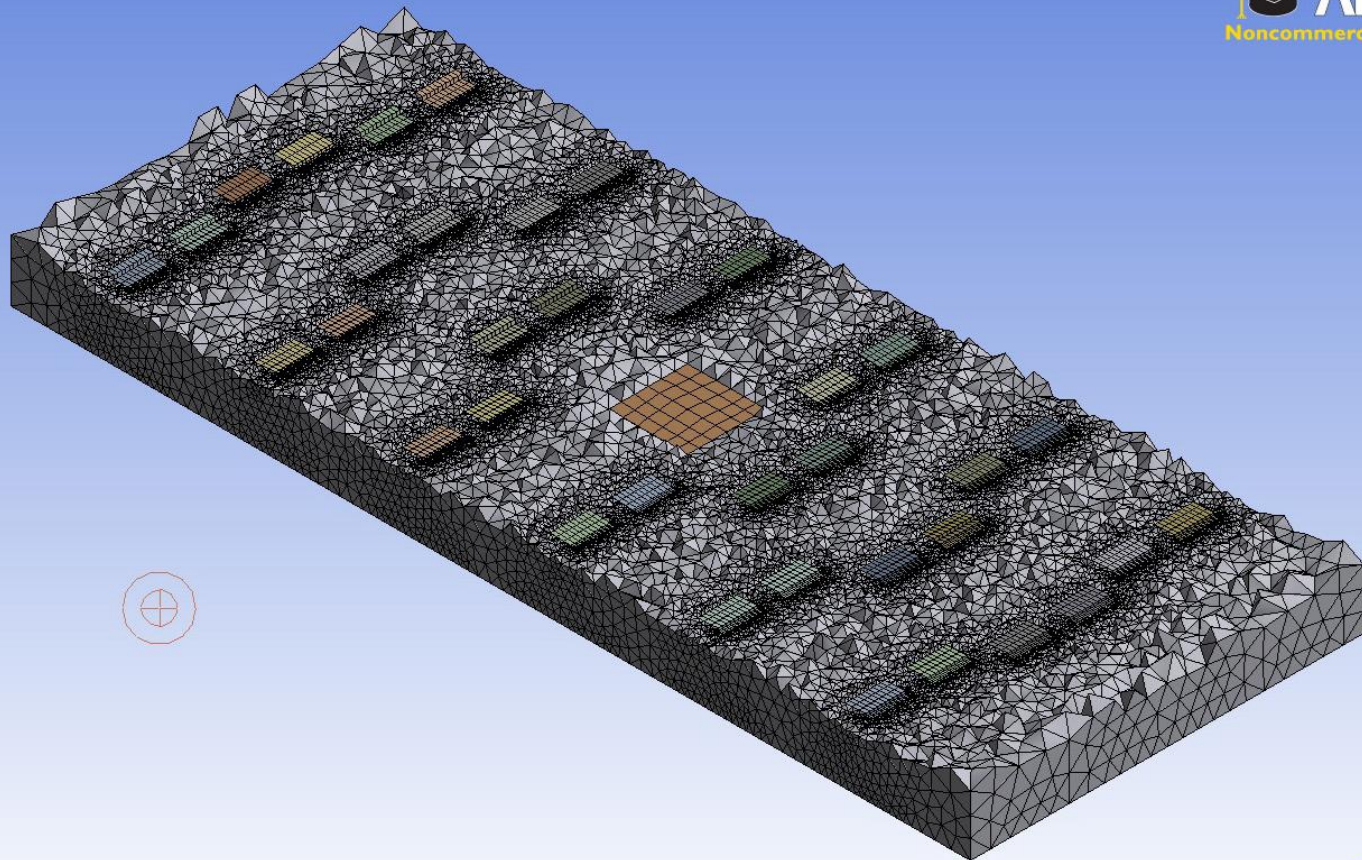
High Seam Refuge Chamber Geometry



Low Seam Refuge Chamber Geometry

Model Design

	High Seam Refuge Chamber	Low Seam Refuge Chamber
Length	11.2 m	11.2 m
Width	4.6 m	4.6 m
Height	1.2 m	0.8 m
Number of Occupants	36	36
Floor Space per Occupant	1.4 m ²	1.4 m ²
Volume per Occupant	1.7 m ³	1.1 m ³
Heat Production per Occupant	117 W	117 W
Heat From Scrubber (per Occupant)	52 W	52 W



Mesh for the High Seam Refuge Chamber

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Model Design

- ◆ **Heat flux from a wall to the adjacent fluid cell calculated as:**

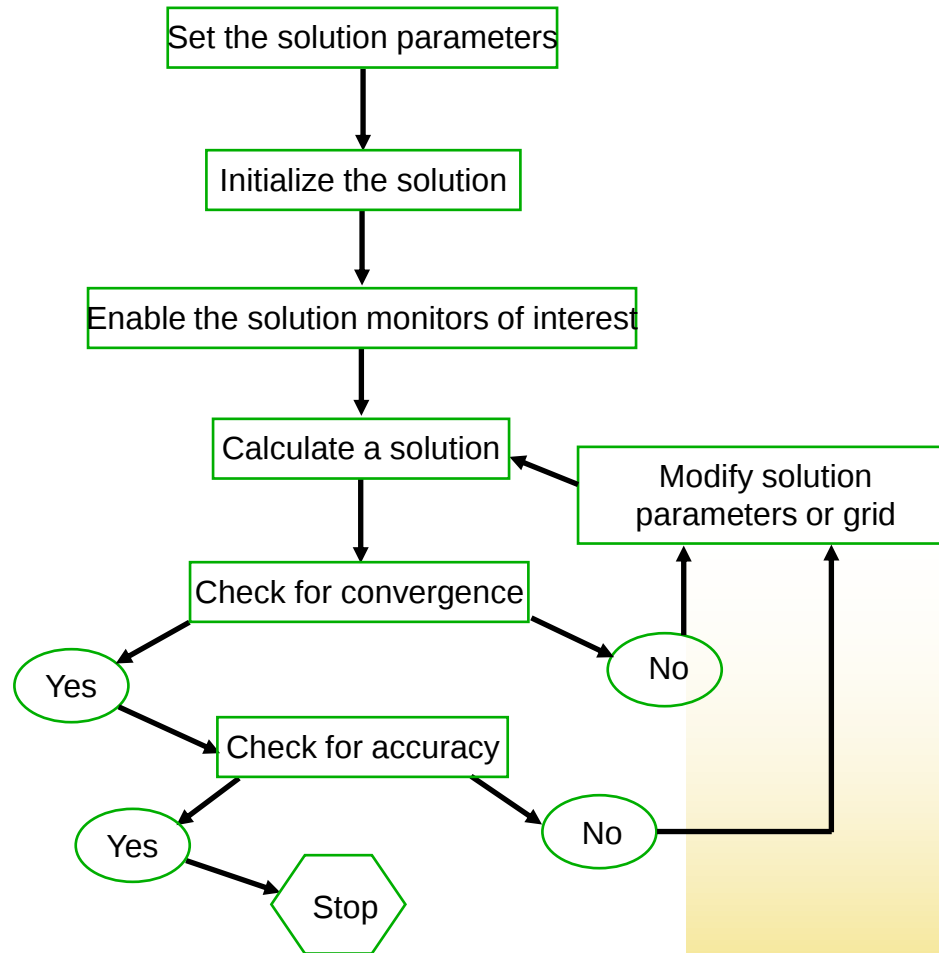
$$q = h_f(T_w - T_f) + q_{rad}$$

- ◆ **Fluid side heat transfer coefficient (h_f) determined based on the local flow properties**

Model Design

- ◆ **For realistic heat transfer coefficients, model must correctly resolve buoyancy driven flow**
- ◆ **Realizable kappa-epsilon (κ - ϵ) model used for turbulence closure**
- ◆ **Incompressible ideal gas law used to model density variation:**

$$\rho = \frac{P_{op}}{\frac{R}{M_w} T}$$



Model Design

Three different structural materials were modeled

- 1) Steel**
- 2) Rubber**
- 3) Aluminum**

Six different external temperatures were considered

- 1) 16° C (60° F)**
- 2) 16.7° C (62.1° F)**
- 3) 21° C (70° F)**
- 4) 27° C (80° F)**
- 5) 32° C (90° F)**
- 6) 38° C (100° F)**

Model Design

	Steel Refuge Chamber	Aluminum Refuge Chamber	Rubber Refuge Chamber
Density	7,833 kg/m ³	2,739 kg/m ³	1,100 kg/m ³
Thickness	6.4 mm	6.4 mm	1.8 mm
Specific Heat	502 J/kg●K	896 J/kg●K	2,010 J/kg●K
Thermal Conductivity	45.3 W/m●K	222 W/m●K	0.13 W/m●K
Emissivity	0.90	0.90	0.86

**The emissivity value for the steel and aluminum chambers assumes a painted surface*

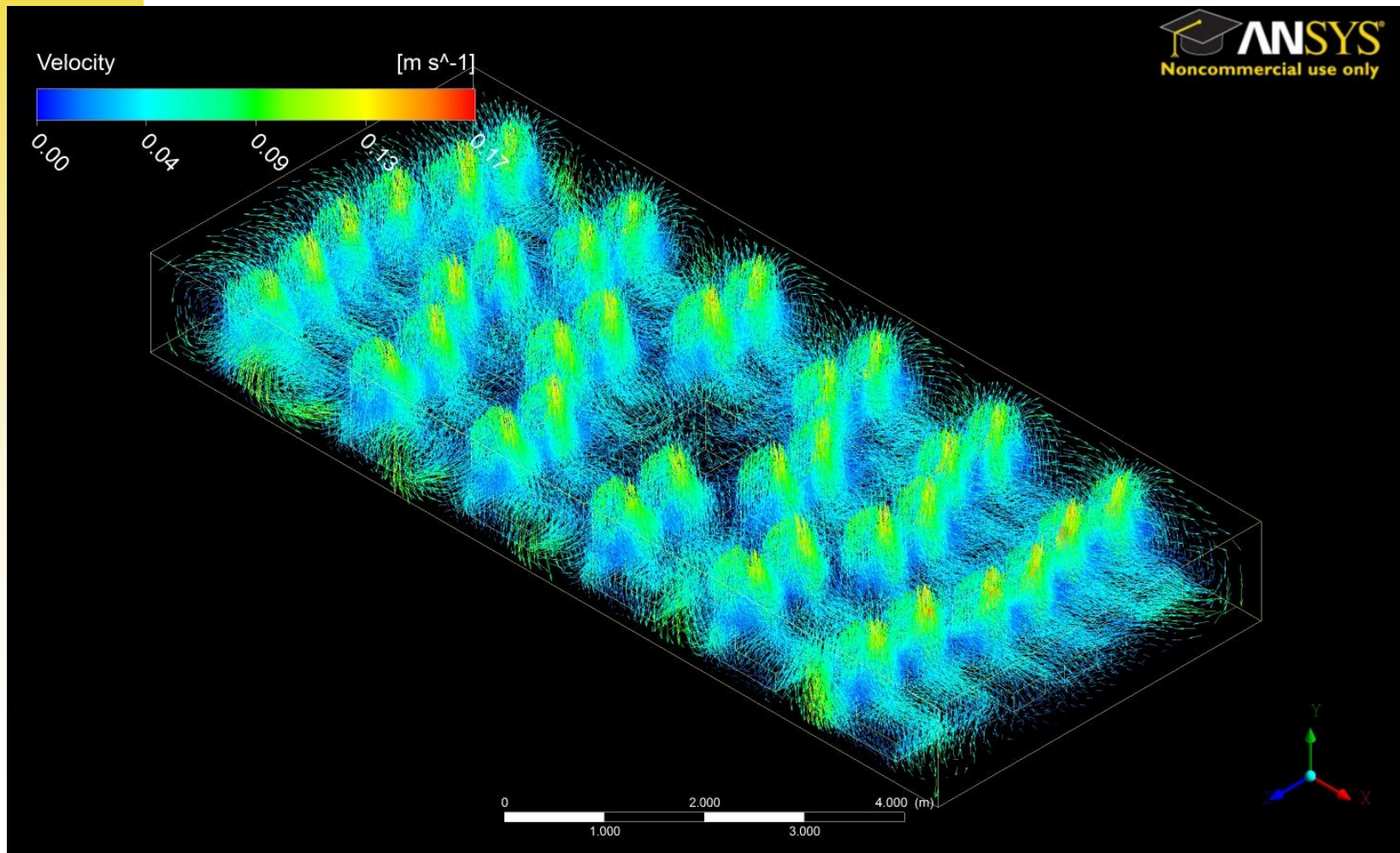
Model Design

Assumptions:

- 1) Temperature outside the chamber was assumed to be constant**
- 2) No cross air flow over the chamber**
- 3) Convective heat transfer coefficient of $5 \text{ W/m}^2\cdot\text{K}$ for the outside of the chamber**
- 4) The mine floor was assumed to be at a constant temperature**
- 5) Perfect thermal contact between chamber bottom and the mine floor**

Results

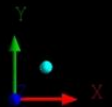
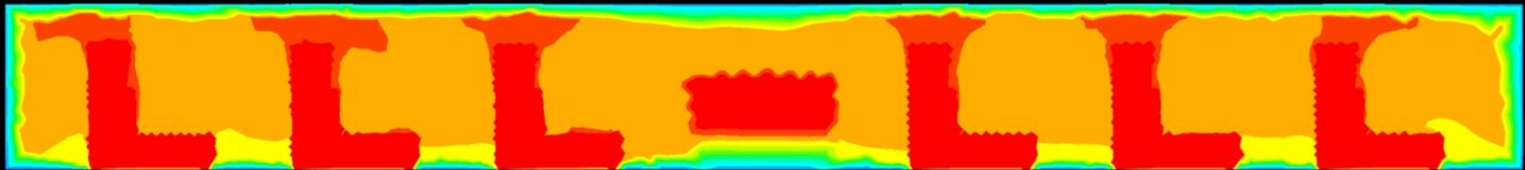
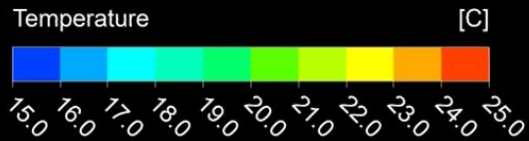
- ◆ **NIOSH in-mine testing validates our simulation results**
- ◆ **Average internal temperature for our generic refuge chamber was 24.7° C (76.5° F)**
- ◆ **Average internal temperature for a refuge chamber of similar design in NIOSH testing was 25.3°C (77.5° F)**
- ◆ **Natural convection currents apparent in model results**



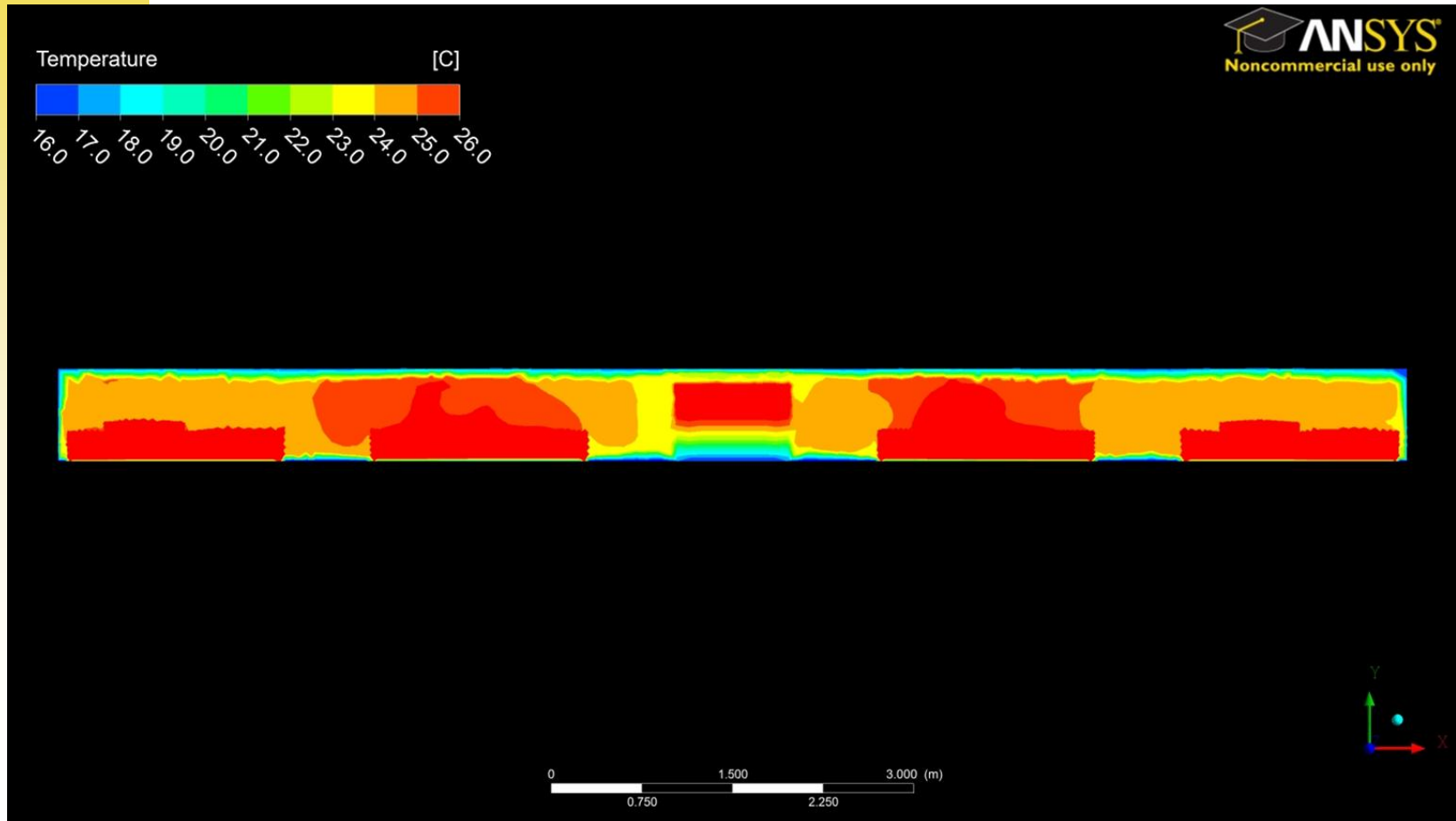
Velocity Vectors for the High Seam Refuge Chamber (External Temp. 16° C)

Results—Impact due to Chamber Size

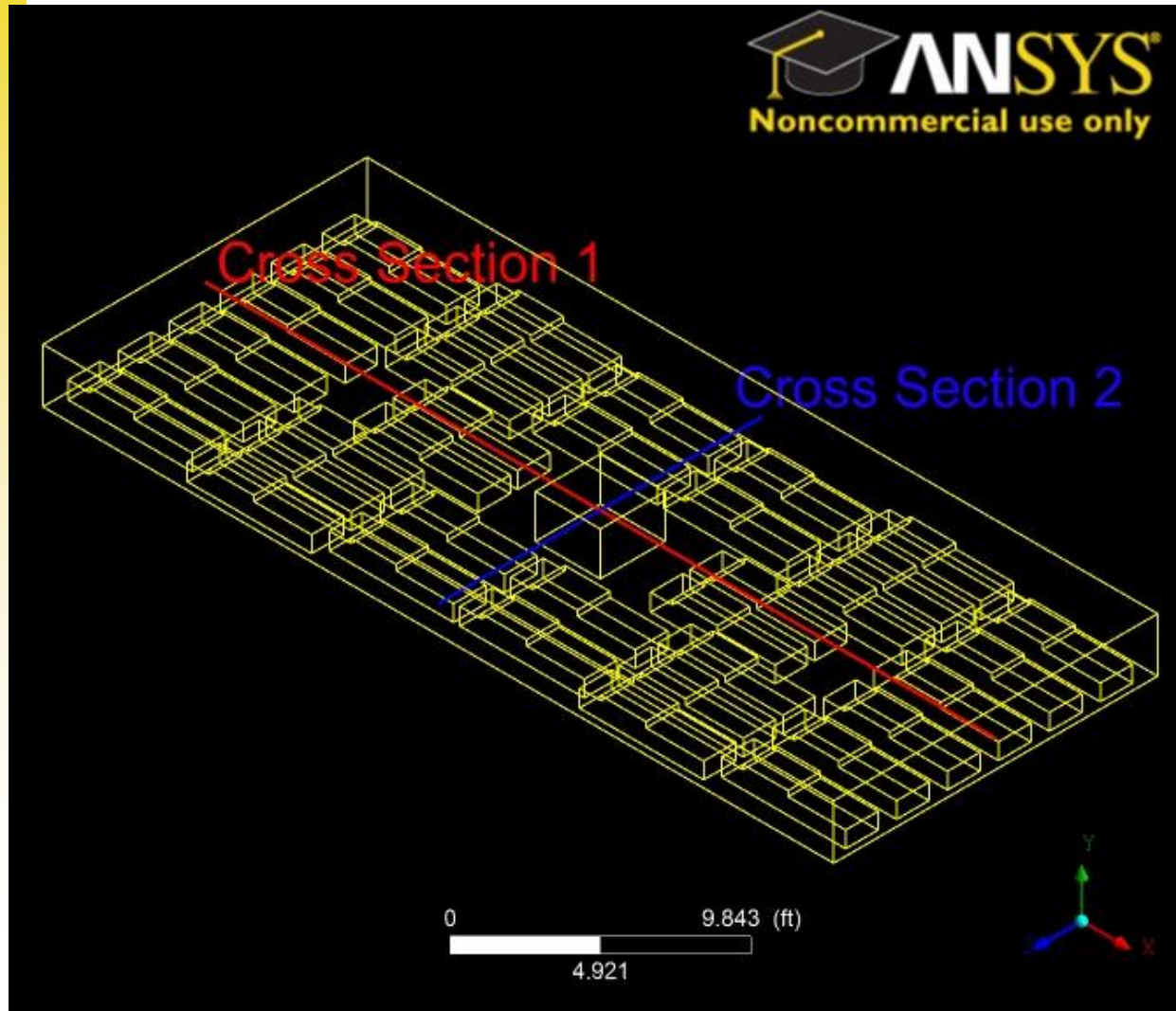
- ◆ **Chamber size does influence internal temperature**
- ◆ **At an external temperature of 16° C (60° F), the high seam steel chamber had an average internal temperature of 23.6° C (74.5° F)**
- ◆ **The low seam steel chamber had an average internal temperature of 24.5° C (76.0° F)**



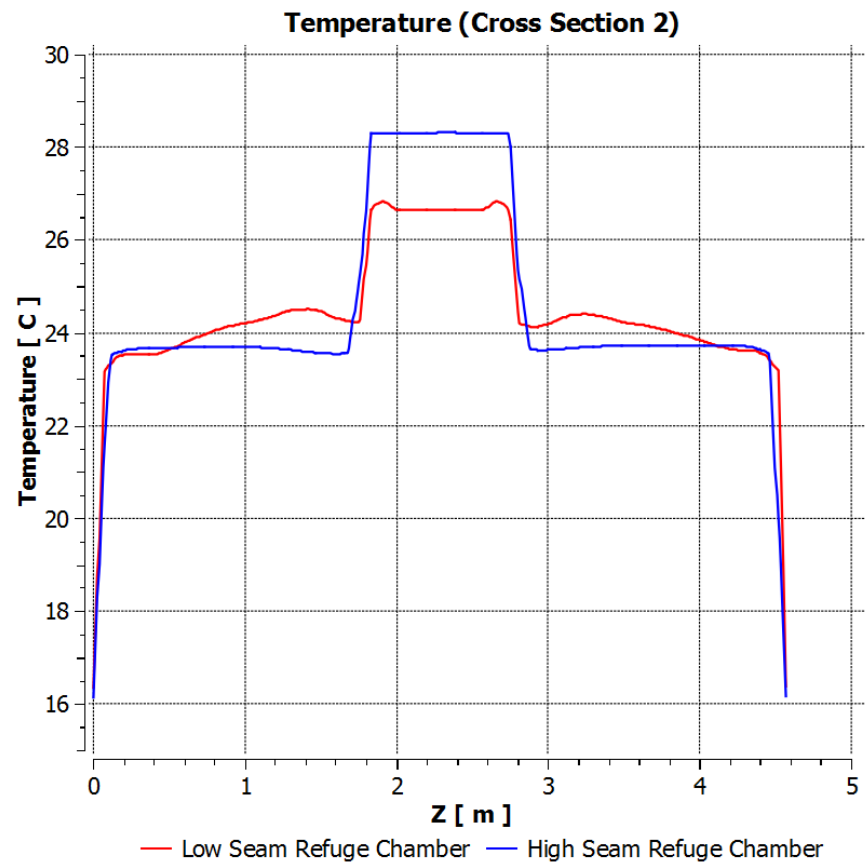
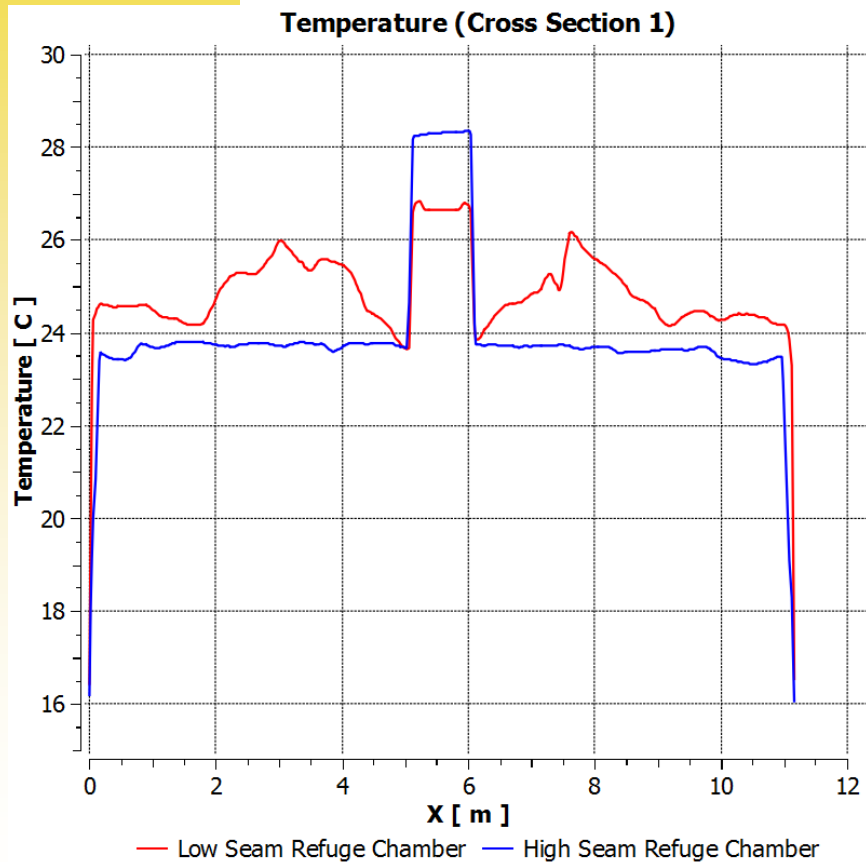
Temperature Contours for the High Seam Chamber (External Temp. 16° C)



Temperature Contours for the Low Seam Chamber (External Temp. 16° C)

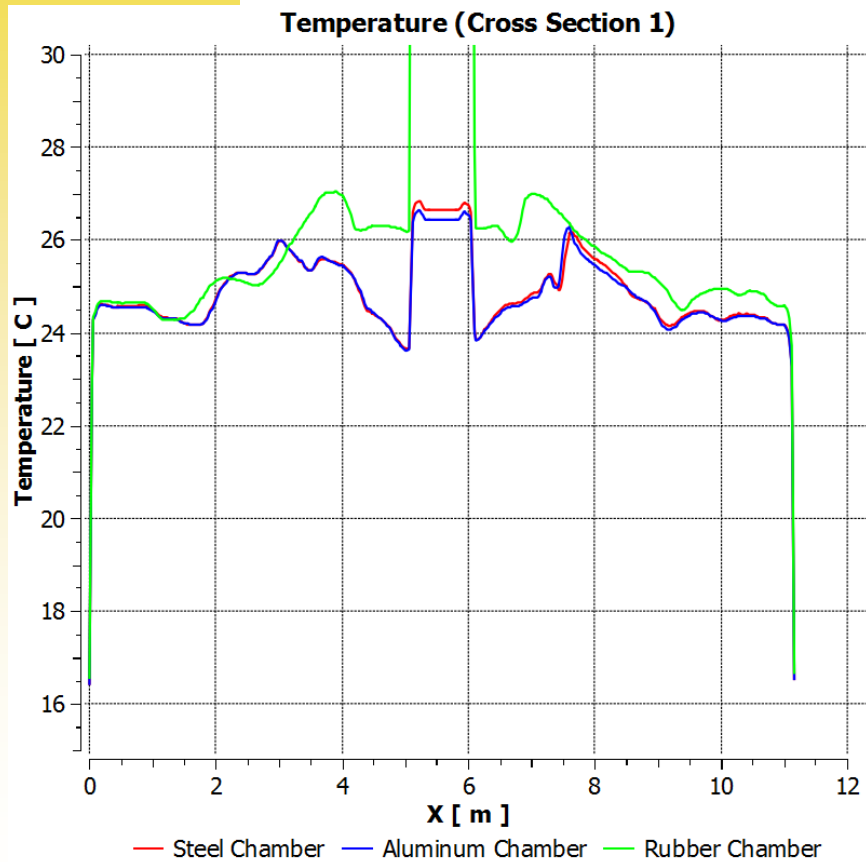


Cross Sections for Temperature Plots

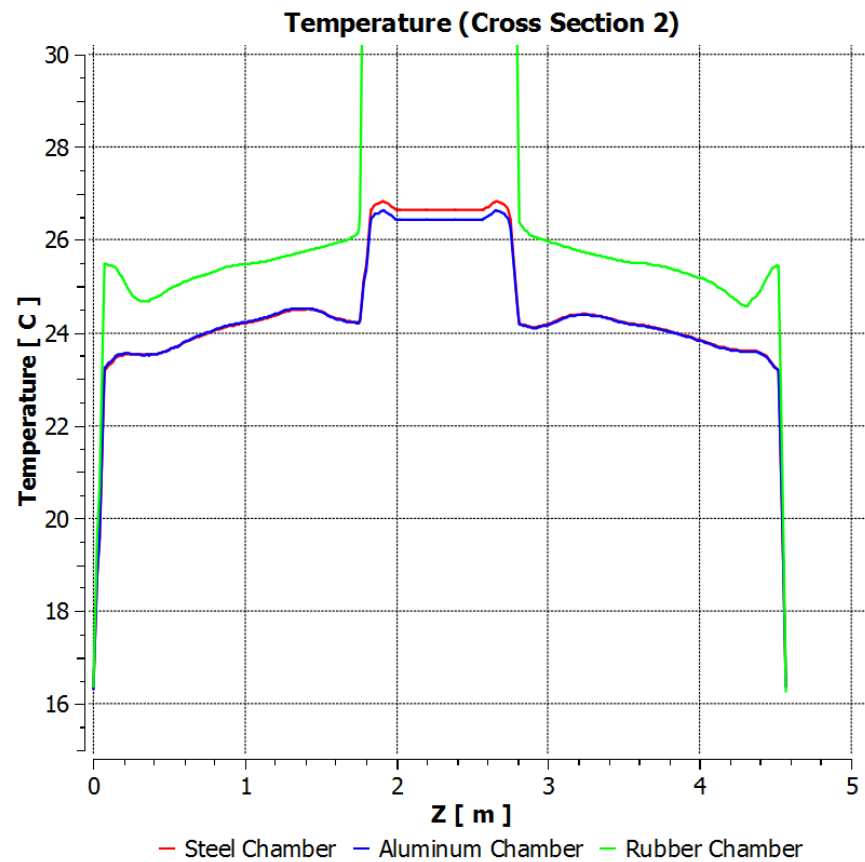


Results—Impact due to Chamber Structure

- ◆ **The structural material of the chamber does impact the internal temperature**
- ◆ **Conduction resistance is insignificant**
- ◆ **Radiation resistance is important**
- ◆ **Temperature inside steel and aluminum chamber is virtually identical**
- ◆ **Rubber chamber is approximately 0.5°C (1°F) hotter on average**



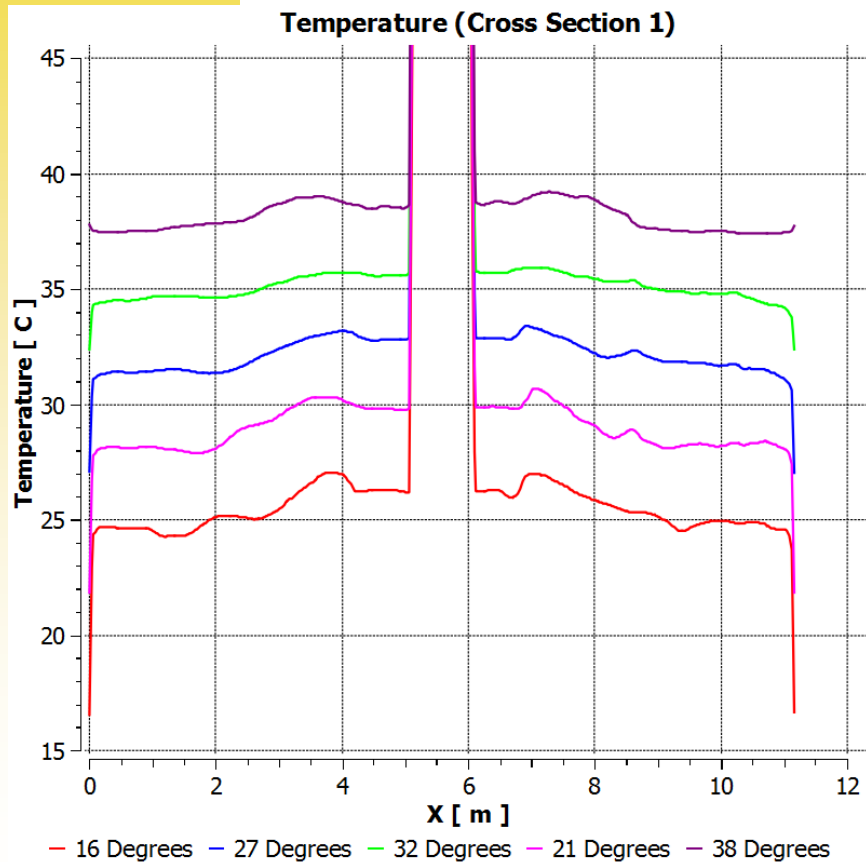
Internal Temperature Along Cross Section 1



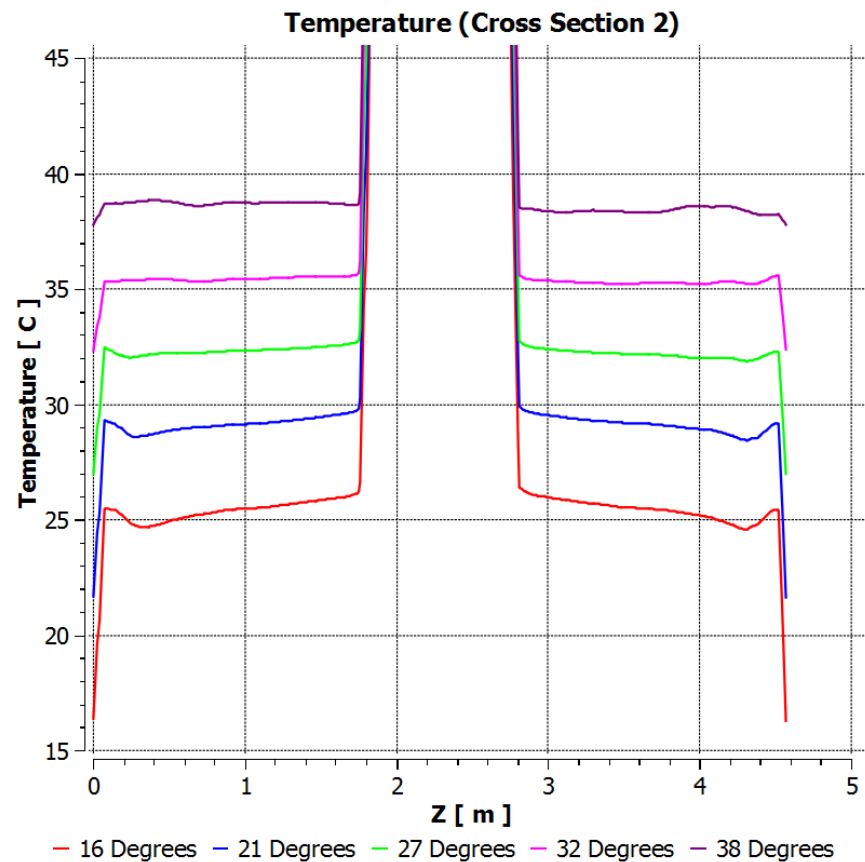
Internal Temperature Along Cross Section 2

Results—Effect of External Temperature

- ◆ **The rubber, low seam chamber was used**
- ◆ **For saturated air, a temperature of 28° C (82° F) exceeds the Steadman Heat Index criteria**
- ◆ **An average internal temperature of 38.3° C (101.0° F) was observed for an external temperature of 38° C (100° F)**
- ◆ **Heat stress a major concern**



Internal Temperature Along Cross Section 1



Internal Temperature Along Cross Section 2

Conclusions

- ◆ **Simulation is a valuable tool for modeling refuge alternatives**
- ◆ **Size has some impact on the internal temperature of the chamber**
- ◆ **Structural materials plays a small role in determining the internal temperature of the chamber**
- ◆ **Heat stress would become a serious concern as external temperatures increase**

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- ◆ **We also wish to thank the University of Alaska Fairbanks for partial travel funding to attend**